

RESISTANCE of MATERIALS

By FRED B. SEELY, M.S.

Professor of Theoretical and Applied Mechanics
University of Illinois

THIRD EDITION

NEW YORK: JOHN WILEY & SONS, INC.

LONDON: CHAPMAN & HALL, LIMITED

Copyright, 1925, 1935, 1947 BY FRED B. SEELY

All Rights Reserved

This book or any part thereof must not be reproduced in any form without the written permission of the publisher.

RESISTANCE of MATERIALS

WORKS OF FRED B. SEELY, M.S. Published By

JOHN WILEY & SONS, INC.

RESISTANCE OF MATERIALS Second Edition. 486 pages. 55% by 85%. 439 figures. Cloth.

ADVANCED MECHANICS of MATERIALS

xi + 331 pages. 6 by 9½. 242 figures. Cloth.

By F. B. SEELY and N. E. ENSIGN ANALYTICAL MECHANICS for ENGINEERS

Third Edition, Rewritten. xv + 450 pages. 6 by $9\frac{1}{4}$. 612 figures. Cloth.

PREFACE TO THE THIRD EDITION

In the third edition of this book, as in the previous editions, the main problem dealt with consists of determining the relationships between the loads acting on a member and the resulting stresses and deformations in the member. Throughout the book the aim has been to give strong emphasis to the engineering significance of the subject, and not merely to give emphasis to mathematical methods of analyses.

This book is concerned primarily with the elementary or basic knowledge of the subject of mechanics of materials or resistance of materials, but the nature of basic knowledge in the subject gradually changes with the development of new methods and materials resulting from applications to new needs and service conditions. Changes in the treatment of the subject in this third edition resulting from relatively recent trends, however, have been introduced only to the extent that seemed feasible. It may be well to mention several of the trends that have gained in importance during the twelve years that have elapsed since the second edition of this book appeared, even though they can be introduced in an elementary treatment only to a minor degree.

Lightweight metals, such as aluminum and magnesium, and plastics have become more widely used in load-resisting members, especially in members in which the strength-weight ratio must be large. This trend accompanied by the increased use of members having thin sections has emphasized the importance of buckling as a mode of failure of members.

Likewise the wider use of high-strength metals such as alloy steels has raised the question of the extent to which higher working stresses should be employed, and especially the question of the optimum advantage that can be taken of such materials in relation to the properties of the material such as yield point, ultimate strength, endurance limit, ductility, etc.

The more extensive use of welded connections and of such practices as shot peening and cold rolling of highly stressed surfaces has introduced residual stresses and multiaxial states of stress as important factors in determining the strength of various members. The increasing use of members under repeated loading as in airplanes, automobiles, railroad rolling stock, and also in railroad tracks and bridges as well as in innumerable types of high-speed equipment has made the problem of stress concentration of extreme importance.

Similarly the use of metals at very low temperatures as occurs in airplanes at great altitudes, in oil-cracking equipment and the like, and the use of metals at very high temperatures as occurs in steam turbines, supercharges, jet-propulsion equipment introduce quite different properties of materials and quite different methods of determining significant stresses from those commonly used in connection with room temperatures.

In the usual problem in resistance of materials, the material is assumed to act only within its elastic range. For certain uses of material, even under ordinary temperatures and static loads, some plastic deformation may occur without damaging the load-carrying capacity of the member, and such plastic deformations may influence the method of calculating the significant stresses. Or to express the idea in other words, some knowledge is needed of the subject of plasticity as well as of elasticity of materials.

The properties of materials which have been employed successfully to indicate satisfactory materials for use at room temperature under static loads and uniformly distributed uniaxial stress may be quite unreliable for indicating whether material is satisfactory for use in another set of conditions involving quite different combinations of temperature, state of stress, distribution of stress, speed of straining, and strain-aging properties of the materials.

Obviously, the foregoing developments and changes in emphasis in the types of problems involving load-resisting members cannot be reflected adequately in an elementary textbook on the mechanics of materials or resistance of materials. In this third edition, however, some attention has been given to a number of these trends with the thought that the instructor or reader can amplify the topics to suit his purposes. This emphasis nevertheless has not changed materially the main topics treated nor the character of the book.

In the third edition, as in the second edition, the book is divided into two parts. Part I consists of the more elementary parts of the subject, and Part II consists of special topics which lie just beyond several of those discussed in Part I. Furthermore, each

topic or chapter in Part II is independent of other topics in Part II so that any chapter in this part may be studied, after the main portion of Part I has been completed, without referring to any other chapter in Part II.

The book has been reset completely; a large number of new problems and new figures have been added. Many minor changes in wording, in arrangement, and in content have been made throughout the book in response to suggestions offered by teachers, students, and practicing engineers who have used the second edition of the book during the past twelve years.

In addition to these minor changes important modifications that have been made may be listed as follows: A decrease in the use made of the double-integration method of determining deflection of beams and a corresponding increase in the use of the areamoment method; an increase in emphasis on the use of Mohr's circle in determining the relation of stresses at a point on different planes passing through the point; a considerable change in the chapter on columns to bring the treatment of this topic more nearly in line with recent reports of technical societies and with present practice; some additional emphasis on stress concentration; and the addition of a brief chapter on the vibrational characteristics on load-resisting members.

The author wishes to express his appreciation of the interest of teachers and others in the second edition of the book as shown by the many comments and helpful criticisms offered.

F. B. SEELY

Urbana, Illinois November 1946

CONTENTS

PART I. ELEMENTARY TOPICS

Art.	Chapter I. Stress and Strain	Page
1	Introduction.	1
2	Types of Loading	3
3	Stresses Due to Central Loads	4
4	Stresses on Oblique Section	11
5	Strains Due to Central Loads	13
6	Stress-strain Curve. Proportional Limit. Yield Point. Ultimate	
	Strength. Elastic Limit. Yield Strength. Modulus of Elasticity	
7	Properties of Structural Materials	24
8	Statically Indeterminate Axial-loaded Members	29
9	Working Stress and Factor of Safety	31
10	Thermal Stresses	36
11	Stress Concentration	39
12 13 14 15 16 17	RIVETED AND WELDED JOINTS Stresses in Thin-walled Cylinders and Spheres	46
19	Welded Joints	60
	*	
	Chapter III. Stress and Strain Caused by Torsional Loads	S
20	Torsional Loads and Twisting Moment	67
21	The Torsion Formula	67
22	Justification for Assumption of Stress Distribution	70
23	Twisting Moment in Terms of Horsepower and Speed	76
24	Angle of Twist of Cylindrical Shaft	78
25	Statically Indeterminate Torsional Member	
26	Stress Beyond Proportional Limit. Modulus of Rupture	
27	Longitudinal Shearing Stress in Torsional Member	
28	Diagonal Tensile (and Compressive) Stress in Torsional Member .	84
29	Torsion of Thin-walled Tubes. Shear Flow	86

Art.	Chapter IV. Transverse Loads. Stresses in Beams	Page
30	Preliminary Considerations	89
31	Vertical Shear, Resisting Shear, Bending Moment, and Resisting	00
01	Moment	92
32	The Flexure Formula	95
33	Section of Maximum Bending Moment	102
34	Shear and Moment Diagrams	103
35	Relation between Shear and Moment	113
36	Overhanging Beams	115
37	Bending Moments Determined Graphically	118
38	Economical Sections of Beams	120
39	Shearing Stress in a Beam	122
40	Stress Beyond Proportional Limit. Modulus of Rupture	129
41	Maximum Moment Due to Moving Loads	132
42	Assumptions and Limitations Involved in the Flexure Formula $$.	134
	Chapter V. Deflection of Beams	
43		137
44	Introduction	137
45	Double-integration Method	142
46	Deflection of Simple Beam, Uniform Load	143
47	An Alternate Method of Determining the Slope to the Elastic Curve	145
48	Moment-area Method	148
49	Simple Beam; Concentrated Load at Mid-span	151
50	Cantilever Beam; Concentrated Load at End	153
51	Simple Beam; Cross Section Not Constant	154
52	Simple Beam; Uniformly Distributed Load	155
53	Simple Beam; Concentrated Load Not at Mid-span	157
54	General Expression for Maximum Deflection	161
55	Relations Among Load, Shear, Moment, Slope, and Deflection .	162
5 6	Deflection of Beam Due to Shear	163
	CHAPTER VI. STATICALLY INDETERMINATE BEAMS	
57	Introduction	165
58	Double-integration Method	165
59	Beam Fixed at One End, Supported at Other End; Uniform Load.	167
60	Moment-area Method	171
61	Beam Fixed at Both Ends; Concentrated Load at Mid-span	172
62	Beam Fixed at Both Ends; Load Uniformly Distributed	175
63	Beam Fixed at One End, Supported at Other End; Load Concen-	
	trated at Mid-span	176
64	Continuous Beam. Theorem of Three Moments	180
65	Solution of Typical Problem	182
66	Values of Moments and Shears	184
67	Theorem of Three Moments for Concentrated Loads	186
68	Comparison with Simple Beams	187
69	Maximum Bending Moments and Maximum Deflections for Fixed-	
	anded Raams	199

	CHAPTER VII. COMBINED AXIAL AND BENDING LOADS.	
Art.	ECCENTRIC LOADS	Page
70	Introduction	190
	A Beam Subjected to an Axial End Load	190
$\frac{71}{72}$	Eccentric Longitudinal Load in Plane of Symmetry	195
		201
73	Eccentric Loads on Riveted Connections	201
74	Helical Spring	204
	CHAPTER VIII. COMPRESSION MEMBERS. COLUMNS	
75	Introduction	208
76	Two Limiting Cases of Compression Members. Two Modes of	
	Failure	209
77	Columns with Intermediate Slenderness Ratios	212
78	Straight-line Formula	214
79	End Conditions	218
80	Straight-line Formula for Restrained Ends	219
81	Gordon-Rankine Column Formula	221
82	The Parabolic Formula	226
83	Comparison of Formulas for Working Loads	228
84	Euler's Column Formula for the Critical Buckling Load	229
85	Effect of End Conditions on Ideal Slender Columns	234
86	Rational Column Formulas	236
87	Secant Formula	237
88	Eccentrically Loaded Columns	244
	CHAPTER IX. RELATIONS BETWEEN STRESSES AT A POINT ON	
	DIFFERENT PLANES PASSING THROUGH THE POINT.	
	THEORIES OF FAILURES	
89	Introduction	248
90	Maximum Shearing Stress in Terms of Principal Stresses	249
91	Mohr's Circle; Principal Stresses Given	253
92	Shearing Stresses Combined with One Normal Stress	255
93	Mohr's Circle; Shearing Stresses Combined with One Normal	
	Stress	265
94	Diagonal Tensile Stress in a Beam	267
95	Strain Due to Principal Stresses. Expression for E_{ϵ}	269
96	Theories of Failure of Elastic Action and Their Application	271
	CHAPTER X. REPEATED LOADS. FATIGUE OF METALS	
97	Introduction	278
98	Localized Stress and Fatigue Failure	280
99	Method of Calculating Localized Stress. Stress-concentration	
	Factor	282
100	Allowable Stress and the Endurance Limit	287
101	Effect of Range of Stress	297
102	Corrosion Fatigue	301
103	Methods of Reducing Harmful Effects of Stress Concentration .	302

Art.	CHAPTER XI. IMPACT AND ENERGY LOADS	Page
104	Introduction	304
104	Calculation of Energy Delivered to Resisting Member	305
106	Stress in Bar Due to Axial Energy Load	306
107	Comparison of Effects of Static and Energy Loads	307
108	Special Cases of Axial Energy Loads	310
109	Ultimate Energy Resistance of a Material. Toughness	314
110	Working Stress and Working Value of Energy	317
111	Stresses in Beams Due to Energy Loads	319
112	Effect of Form on Energy Resistance of Beams	322
113	Special Cases of Energy Loads on Beams	323
114	Torsional Energy Load	326
115	Reduction in Energy Load Due to Inertia of Resisting Member .	327
	PART II. SPECIAL TOPICS	
116	Explanation of Part II	329
	CHAPTER XII. CONJUGATE-BEAM METHOD	
117	Conjugate Beam Defined	331
	STATICALLY DETERMINATE BEAMS	
118	Simple Beam; Concentrated Load at Mid-span	334
119	Cantilever Beam; Concentrated Load at End	335
120	Simple Beam; Load Distributed Uniformly	336
121	Cantilever Beam; Load Distributed Uniformly.	337
122	Simple Beam; Concentrated Load at Any Point	337
123	Simple Beam; Cross Section not Constant	338
	STATICALLY INDETERMINATE BEAMS	
124	Beam Fixed at Both Ends; Load Distributed Uniformly	340
125	Beam Fixed at Both Ends; Concentrated Load at Mid-span	341
126	Beam Fixed at One End; Supported at Other End; Load Uni-	
	formly Distributed	342
127	Continuous Beam; Theorem of Three Moments	343
	CHAPTER XIII. LIMITATIONS OF THE FLEXURE FORMULA FOR SECTIONS HAVING ONLY ONE AXIS OF SYMMETRY	9
128	Bending Axis and Shear Center Defined	346
129	The Shear Center and the Flexure Formula	346
130	Approximate Location of the Shear Center for a Channel Section.	347
131	Shear Center for Certain Other Sections Having One Axis of Sym-	
	metry	352
	CHAPTER XIV. UNSYMMETRICAL BENDING	
132	Introduction	355
133	Stress in Beam Subjected to Unsymmetrical Loading	

	CONTENTS	xiii
Art.		Page
134	Change in Slope of Neutral Axis and Increase in Stress in Rolled Sections Due to a Very Small Inclination of Plane of Loads to a Principal Plane	362
135	Eccentric Load Not in Plane of Symmetry	363
	CHAPTER XV. CURVED FLEXURAL MEMBERS	
136	Introduction	366
137	Essential Difference between Straight Beams and Curved Beams.	366
138	Unit Stress at Any Point in a Curved Beam. The Winkler-Bach Formula	368
139	Correction Factors for Use in the Straight-beam Formula	372
100	Correction Pactors for ose in the Straight Seam Formula.	
	CHAPTER XVI. FLAT PLATES	.1.
140	Introduction	377
141	Stress in Circular Plate	377
142	Stress in Square Plate	379
143	Stress in Rectangular Plate	382
	CHAPTER XVII. THICK-WALLED CYLINDERS	
144	The Problem Defined	388
145	Lamé's Solution	388
146	Maximum Shearing Stress	394
147	Maximum Strain and Maximum Value of E_{ϵ}	396
	Chapter XVIII. Torsional Resistance of Bars Having Noncircular Cross Sections	
148	Introduction	398
149	Introduction	
150	Section	399
150 151	Elastic Membrane (Soap-film) Analogy	401 404
101	Stress and Angle of Twist for Several Sections	404
	Chapter XIX. Composite Beams. Reinforced-concrete Beams	
152	Composite (Two-material) Beam	406
153	Timber Beam Reinforced by Steel Plates	406
154	Reinforced-concrete Beams	409
155	Stress in Steel and in Concrete	410
156	T Beams	414
157	Beams Reinforced in Both Tension and Compression	417
158	Shearing Stress	419
159	Bond Stress and Anchorage	420
160	Diagonal Tension	421
161	Reinforcement for Diagonal Tension	422

Part I. Elementary Topics

CHAPTER I

STRESS AND STRAIN

1. Introduction. Resistance of materials is that branch of mechanics which treats of the internal forces in a physical body and of the changes of shape and size of the body, particularly in their relation to the external forces that act on the body and to the physical properties of the material of the body. The external forces that act on the body are called *loads*; the internal forces, which resist the external forces, are called *stresses*, and the accompanying changes in the dimensions of the body are called *deformations* or *strains*.

The total stress ¹ on a section through a body is the total internal force acting on the section; that is, if the body is thought of as being divided in two parts by the section, the total stress at the section is the force exerted by one part of the body on the other part in order to hold in equilibrium the loads acting on the part considered. The component of the internal force normal to the area is called *normal stress*, and the component tangent to (or in) the area is called *shearing stress*. Further, a normal stress may be a tensile stress or a compressive stress according to whether the body is stretched or shortened.

Intensity of stress or unit stress 1 is defined to be internal force per unit of area. In general the unit stress varies from point to point over a section, its value at any point being considered to be the internal force on an elementary or differential part of the area, including the point, divided by the elementary area; but when the stress is distributed uniformly on an area the unit stress at all points in the section is equal to the total force on the area divided by the whole area. It is important to note that, in general,

¹ In technical literature the term stress sometimes is used to denote what is here defined as intensity of stress or unit stress, and the term internal force is used to denote what is here called total stress.

a unit stress occurs at a point in a body on some plane passing through the point, and that the most significant unit stresses at the point on the plane considered are the normal and shearing stresses, since these stresses are most closely associated with damage to the material.

A unit stress may be expressed in various units, such as pounds per square inch (lb. per sq. in. or lb./in.²; the abbreviation psi. also is used), kilograms per square centimeter (kg. per sq. cm.), tons per square inch (ton per sq. in.), etc.

The total deformation or total strain in any direction of a body subjected to loads is the total change in the dimension of the body in that direction, and the unit deformation or unit strain in any direction is the deformation or strain per unit of length in that direction

The Problem Defined. A body when subjected to loads is stressed and deformed, and the values of the stresses and the deformations in the body are of great importance in many engineering problems. These stresses are found by use of the general principles of mechanics (mainly of statics) and of the experimental laws that have been found to govern the action of the material. The main objectives, then, in the study of resistance of materials are:

- (1) To determine the relation between the external forces (loads) acting on a body and the unit stress at some point in the body on some plane passing through the point, so that the unit stresses caused by known loads may be calculated. The point and plane selected, in general, will be the ones that give the stress which would first cause damage to the material if the loads were increased.
- (2) To determine the relation between loads acting on a body and the resulting strains produced in the body so that strains may be determined from known loads, or vice versa.
- (3) To obtain a knowledge of the physical properties, such as strength, stiffness, ductility, toughness, resilience, of the various structural materials, since physical properties of the materials are involved in the relations under (1) and (2).

As suggested by the statements under (1) and (2), in some problems of design the stress produced in the body by the load is the governing factor in the design, whereas in other problems the strain (or elastic deflection) produced is the more important factor. For example, the chain of a hoist fulfills its function of lifting loads regardless of the amount of stretch of the chain that occurs, provided that the load does not produce a stress large

enough to cause excessive permanent distortion or to rupture the chain. That is, the stress developed in the chain by the maximum load to be applied is the governing factor in the design of the chain rather than the elastic deformation of the chain. Likewise, stress is the governing factor in the design of many of the parts of buildings, cranes, ships, etc. On the other hand, the maximum load that some members of structures and machines can resist satisfactorily is limited by the permissible elastic deflection of the member rather than by the permissible maximum stress in the member. For example, machine tools such as lathes, drill presses, and boring mills will not produce work of sufficient accuracy if the elastic deformations of the parts are too large.

- 2. Types of Loading. With reference to the manner in which loads are applied or transmitted to a structure or machine, the loads will be considered under three distinct headings: namely, static loads, repeated loads, and impact and energy loads.
- 1. Static, steady, or dead loads are forces that are applied slowly and not repeated, and remain nearly constant after being applied to the body, or are repeated relatively few times, such as the loads on most buildings or the load applied to a bar in the usual type of testing machine (see Fig. 3).
- 2. Repeated loads are forces that are applied a very large number of times causing a stress in the material that is continually changing, usually through some definite range. The loads applied to the connecting rod of an engine when the engine is running and the wheel loads on a railroad rail as a train passes over the rail are repeated loads. Repeated loads are discussed in Chapter X.
- 3. Impact loads are forces that are applied to the resisting body in a relatively short period of time. An impact load, in general, is applied by a body that is in motion when it comes in contact with the resisting body, and the force exerted by the moving body and the period during which it acts in general cannot be determined. For this reason in some problems it is more satisfactory to calculate the stress and strain produced by an impact load from the energy delivered to the resisting body by the moving body. When this is done the energy delivered to the resisting body is called an energy load and is expressed in foot-pounds (not in pounds). Impact and energy loads are considered in Chapter XI.

Other Classifications of Loads. Loads may be classified as distributed loads and concentrated loads. A distributed load may be uniformly distributed or non-uniformly distributed. Thus, if sand

is spread on a floor so that its depth is constant, the floor is subjected to a uniformly distributed load, whereas, if the sand is distributed so that its depth is not constant, the floor is said to carry a non-uniformly distributed load. A concentrated load is one whose area of contact with the resisting body is negligible in comparison with the area of the resisting body.

The more common types of loads to which structural and machine members are subjected may be classified as *central* loads, *torsional* loads, and *bending* loads. A body may be subjected simultaneously to loads of any two or to all three of these types, however.

The stresses and strains caused by static central loads are discussed in this chapter. The stresses and strains caused by static torsional loads are considered in Chapter III, and those produced by static bending loads in Chapter IV.

Static Central Loads

3. Stresses Due to Central Loads. A central load is a concentrated load whose action line passes through the centroid of the area on which the stresses are to be considered, or a distributed load whose resultant passes through the centroid of the area. The stress produced by a central load may be any one of the three types, tensile stress, compressive stress, or shearing stress, but the distinguishing feature of a central load that acts on a straight bar of constant cross section as compared to torsional and bending loads is that the stress in the bar may be assumed to be uniformly distributed over the cross-sectional area; that is, the intensity of stress (unit stress) may be assumed to be constant. If a central load acts in a direction normal to the area on which the stress is to be found, it is called an axial load and it may be a tensile or a compressive axial load according to whether it produces tensile or compressive stress; if the central load lies in the plane of the area it is called a shearing central load. The compressive stress exerted by one body on another (different) body at the area of contact usually is called a bearing stress.

Stresses due to central loads may be found as follows: Fig. 1a represents a straight bar AB subjected to an axial tensile load P causing tensile stress on any section of the bar perpendicular to the loads; Fig. 1c represents a bar in compression under the action of an axial load P causing compressive stress in the bar; and