

# INTRODUCTION TO HEAT TRANSFER



Second Edition



FRANK P. INCROPERA DAVID P. DE WIFT



#### SECOND EDITION

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#### FRANK P. INCROPERA DAVID P. DEWITT

School of Mechanical Engineering Purdue University



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Dedicated to those wonderful women in our lives,

Amy, Andrea, Debbie, Donna, Jody, Karen, Shaunna, and Terri

who, through the years, have blessed us with their love, patience, and understanding.

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# INTRODUCTION TO HEAT TRANSFER

#### **PREFACE**

With the passage of approximately nine years since publication of the first edition, this text has been transformed from the status of a newcomer to a mature representative of heat transfer pedagogy. Despite this maturation, however, we like to think that, while remaining true to certain basic tenets, our treatment of the subject is constantly evolving.

Preparation of the first edition was strongly motivated by the belief that, above all, a first course in heat transfer should do two things. First, it should instill within the student a genuine appreciation for the physical origins of the subject. It should then establish the relationship of these origins to the behavior of thermal systems. In so doing, it should develop methodologies which facilitate application of the subject to a broad range of practical problems, and it should cultivate the facility to perform the kind of engineering analysis which, if not exact, still provides useful information concerning the design and/or performance of a particular system or process. Requirements of such an analysis include the ability to discern relevant transport processes and simplifying assumptions, identify important dependent and independent variables, develop appropriate expressions from first principles, and introduce requisite material from the heat transfer knowledge base. In the first edition, achievement of this objective was fostered by couching many of the examples and end-of-chapter problems in terms of actual engineering systems.

The second edition was also driven by the foregoing objectives, as well as by input derived from a questionnaire sent to over 100 colleagues who used, or were otherwise familiar with, the first edition. A major consequence of this input was publication of two versions of the book, Fundamentals of Heat and Mass Transfer and Introduction to Heat Transfer. As in the first edition, the Fundamentals version included mass transfer, providing an integrated treatment of heat, mass and momentum transfer by convection and separate treatments of heat and mass transfer by diffusion. The Introduction version of the book was intended for users who embraced the treatment of heat transfer but did not wish to cover mass transfer effects. In both versions, significant improvements were made in the treatments of numerical methods and heat transfer with phase change.

In this latest edition, changes have been motivated by the desire to expand the scope of applications and to enhance the exposition of physical principles. Consideration of a broader range of technically important problems is facilitated by increased coverage of existing material on thermal contact resistance, fin performance, convective heat transfer enhancement, and

compact heat exchangers, as well as by the addition of new material on submerged jets (Chapter 7) and free convection in open, parallel plate channels (Chapter 9). Submerged jets are widely used for industrial cooling and drying operations, while free convection in parallel plate channels is pertinent to passive cooling and heating systems. Expanded discussions of physical principles are concentrated in the chapters on single-phase convection (Chapters 7 to 9) and relate, for example, to forced convection in tube banks and to free convection on plates and in cavities. Other improvements relate to the methodology of performing a first law analysis, a more generalized lumped capacitance analysis, transient conduction in semi-infinite media, and finitedifference solutions.

In this edition, the old Chapter 14, which dealt with multimode heat transfer problems, has been deleted and many of the problems have been transferred to earlier chapters. This change was motivated by recognition of the importance of multimode effects and the desirability of impacting student consciousness with this importance at the earliest possible time. Hence, problems involving more than just a superficial consideration of multimode effects begin in Chapter 7 and increase in number through Chapter 13.

The last, but certainly not the least important, improvement in this edition is the inclusion of nearly 300 new problems. In the spirit of our past efforts, we have attempted to address contemporary issues in many of the problems. Hence, as well as relating to engineering applications such as energy conversion and conservation, space heating and cooling, and thermal protection, the problems deal with recent interests in electronic cooling, manufacturing, and material processing. Many of the problems are drawn from our accumulated research and consulting experiences; the solutions, which frequently are not obvious, require thoughtful implementation of the tools of heat transfer. It is our hope that in addition to reinforcing the student's understanding of principles and applications, the problems serve a motivational role by relating the subject to real engineering needs.

Over the past nine years, we have been fortunate to have received constructive suggestions from many colleagues throughout the United States and Canada. It is with pleasure that we express our gratitude for this input.

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> > West Lafayette, Indiana

## **SYMBOLS**

1	A	area, m <sup>2</sup>	G	irradiation, W/m <sup>2</sup> ; mass velocity,
	$A_c$	cross-sectional area, m2		$kg/s \cdot m^2$
	$A_{ff}$	free-flow area in compact heat	Gr	Grashof number
		exchanger core (minimum cross-	Gz	Graetz number
		sectional area available for flow	g	gravitational acceleration, m/s2
		through the core), m <sup>2</sup>	$g_c$	gravitational constant,
	$A_{\rm fr}$	heat exchanger frontal area, m2		$1 \text{ kg} \cdot \text{m/N} \cdot \text{s}^2 \text{ or}$
	$A_p$	area of prime (unfinned) surface, m2		32.17 ft $\cdot$ lb <sub>m</sub> /lb <sub>f</sub> $\cdot$ s <sup>2</sup>
	$A_{r}$	nozzle area ratio	H	nozzle height, m
	$A_s$	surface area, m <sup>2</sup>	h	convection heat transfer coefficient,
	a	acceleration, m/s2		W/m <sup>2</sup> · K; Planck's constant
	Bi	Biot number	$h_{fg}$	latent heat of vaporization, J/kg
	Bo	Bond number	$h_{\rm rad}$	radiation heat transfer coefficient,
9	C	heat capacity rate, W/K	140	$W/m^2 \cdot K$
	$C_D$	drag coefficient	I	electric current, A; radiation
	$C_{\ell}$	friction coefficient		intensity, W/m <sup>2</sup> · sr
	$C_D$ $C_f$ $C_i$	thermal capacitance, J/K	i	electric current density, A/m2;
-	c	specific heat, J/kg · K; speed of light,		enthalpy per unit mass, J/kg
		m/s	J	radiosity, W/m <sup>2</sup>
3	$c_p$	specific heat at constant pressure,	Ja	Jakob number
	·	J/kg · K	Ĵн	Colburn j factor for heat transfer
3	$c_v$	specific heat at constant volume,	k	thermal conductivity, W/m · K;
		J/kg · K		Boltzmann's constant
	D	diameter, m	L	characteristic length, m
	$D_h$	hydraulic diameter, m	M	mass, kg; number of heat transfer
	E	thermal (sensible) internal energy, J;		lanes in a flux plot; reciprocal of
		electric potential, V; emissive		the Fourier number for finite-
		power, W/m <sup>2</sup>		difference solutions
	Ec	Eckert number	M	molecular weight, kg/kmol
	$\dot{E}_{ m g}$	rate of energy generation, W	m	mass, kg
- 1	$\dot{E}_{ m g}$	rate of energy transfer into a control	m	mass flow rate, kg/s
		volume, W	N	number of temperature increments in
	$\dot{E}_{ m out}$	rate of energy transfer out of control		a flux plot; total number of tubes
	out	volume, W		in a tube bank; number of surfaces
	$\dot{E}_{\mathrm{st}}$	rate of increase of energy stored		in an enclosure
		within a control volume, W	$N_L$ ,	number of tubes in longitudinal and
	e	thermal internal energy per unit mass,	$N_T$	transverse directions
		J/kg; surface roughness, m	Nu	Nusselt number
	F	force, N; heat exchanger correction	NTU	number of transfer units
		factor; fraction of blackbody	P	perimeter, m; general fluid property
		radiation in a wavelength band;		designation
		view factor	$P_L$ ,	dimensionless longitudinal and
	Fo	Fourier number	$P_T$	transverse pitch of a tube bank
	f	friction factor; similarity variable	Pe	Peclet number (RePr)
		name and the second decomposition of the second sec		, ,

n	Prandtl number	Γ	mass flow rate per unit width in film
Pr	pressure, N/m <sup>2</sup>	1	condensation, kg/s · m
P	energy transfer, J	δ	hydrodynamic boundary layer
Q	heat transfer rate, W		thickness, m
q	rate of energy generation per unit	$\delta_t$	thermal boundary layer thickness, m
ġ	volume, W/m <sup>3</sup>	ε	emissivity; porosity of a packed bed;
· · · · ·	heat transfer rate per unit length,		heat exchanger effectiveness
q'	W/m	6.	fin effectiveness
a''	heat flux, W/m <sup>2</sup>	$\epsilon_f$	turbulent diffusivity for heat transfer,
q''	cylinder radius, m	$\epsilon_H$	$m^2/s$
R R	universal gas constant		turbulent diffusivity for momentum
Ra	Rayleigh number	$\epsilon_M$	transfer, m <sup>2</sup> /s
	Reynolds number	20	similarity variable
Re	electric resistance, $\Omega$	η	fin efficiency
$R_e$	fouling factor, m <sup>2</sup> · K/W	$\eta_f$	fin temperature effectiveness
$R_f$	residual for the $m, n$ nodal point	$\theta$	zenith angle, rad; temperature
$R_{m,n}$	thermal resistance, K/W	U	difference, K
$R_t$	thermal contact resistance, K/W	κ	absorption coefficient, m <sup>-1</sup>
$R_{t,c}$	cylinder or sphere radius, m	λ	wavelength, µm
r <sub>o</sub>	cylindrical coordinates		viscosity, kg/s · m
$r, \phi, z$	spherical coordinates	$\mu$ $\nu$	kinematic viscosity, m <sup>2</sup> /s; frequency
r, θ, φ S	shape factor for two-dimensional	V	of radiation, s <sup>-1</sup>
3	conduction, m; nozzle pitch, m;	0	mass density, kg/m <sup>3</sup> ; reflectivity
	plate spacing, m	ρ σ	Stefan-Boltzmann constant; electrical
C	solar constant	U	conductivity, $1/\Omega \cdot m$ ; normal
$S_c$	diagonal, longitudinal, and transverse		viscous stress, N/m <sup>2</sup> ; surface
$S_D, S_L,$	pitch of a tube bank, m		tension, N/m; ratio of heat
$S_T$ St	Stanton number		exchanger minimum cross-sectional
T	temperature, K		area to frontal area
	time, s	Φ	viscous dissipation function, s <sup>-2</sup>
t U	overall heat transfer coefficient,		azimuthal angle, rad
U	W/m <sup>2</sup> · K	φ	stream function, m <sup>2</sup> /s
	mass average fluid velocity	$\psi$ $\tau$	shear stress, N/m <sup>2</sup> ; transmissivity
u, v,	components, m/s	ω	solid angle, sr
V	volume, m <sup>3</sup> ; fluid velocity, m/s	w	solid aligic, si
v	specific volume, m <sup>3</sup> /kg		7
W	width of a slot nozzle, m	Subscrip	nts
ŵ	rate at which work is performed, W	abs	absorbed
We	Weber number	adv	advection
X, Y,	components of the body force per	am	arithmetic mean
Z	unit volume, N/m <sup>3</sup>	b	base of an extended surface;
x, y, z	rectangular coordinates, m	U	blackbody
0.51.0	critical location for transition to	c	cross-sectional; cold fluid
$x_c$	turbulence, m	cr	critical insulation thickness
Y	hydrodynamic entry length, m	cond	conduction
x <sub>fd, h</sub>	thermal entry length, m	conv	convection
$x_{\text{fd}, t}$	thermal entry length, in	CF	counterflow
Greek 1	Lattere	D	diameter; drag
α	thermal diffusivity, m <sup>2</sup> /s; heat	e	excess; emission
a	exchanger surface area per unit	evap	evaporation
	volume, m <sup>2</sup> /m <sup>3</sup> ; absorptivity	f	fluid properties; fin conditions;
β	volumetric thermal expansion	J	saturated liquid conditions
P	coefficient, K <sup>-1</sup>	fd	fully developed conditions
	comment, is		and a company continuous

#### xvi Symbols

g	saturated vapor conditions	s	surface conditions; solid properties
H	heat transfer conditions	sat	saturated conditions
h	hydrodynamic; hot fluid	sky	sky conditions
i	inner surface of an annulus; initial	sur	surroundings
	condition; tube inlet condition;	t	thermal
	incident radiation	tr	transmitted
L	based on characteristic length	v	saturated vapor conditions
1	saturated liquid conditions	x	local conditions on a surface
lm	log mean condition	λ	spectral
M	momentum transfer condition	∞	freestream conditions
m	mean value over a tube cross section		
max	maximum fluid velocity	Superson	rints
0	center or midplane condition; tube	,	
	outlet condition; outer		fluctuating quantity
R	reradiating surface	7.	dimensionless quantity
r, ref	reflected radiation		
rad	radiation	Overbar	
S	solar conditions	=	surface average conditions: time mean
	h i  L l lm m max o  R r, ref rad	H heat transfer conditions h hydrodynamic; hot fluid i inner surface of an annulus; initial condition; tube inlet condition; incident radiation  L based on characteristic length l saturated liquid conditions lm log mean condition M momentum transfer condition m mean value over a tube cross section max maximum fluid velocity o center or midplane condition; tube outlet condition; outer  R reradiating surface r, ref reflected radiation rad radiation	$H$ heat transfer conditions       sat $h$ hydrodynamic; hot fluid       sky $i$ inner surface of an annulus; initial       sur         condition; tube inlet condition; $t$ incident radiation       tr $L$ based on characteristic length $v$ $l$ saturated liquid conditions $x$ Im       log mean condition $\infty$ $M$ momentum transfer condition $\infty$ $m$ mean value over a tube cross section         max       maximum fluid velocity       Superscription $o$ center or midplane condition; tube $r$ $v$ <

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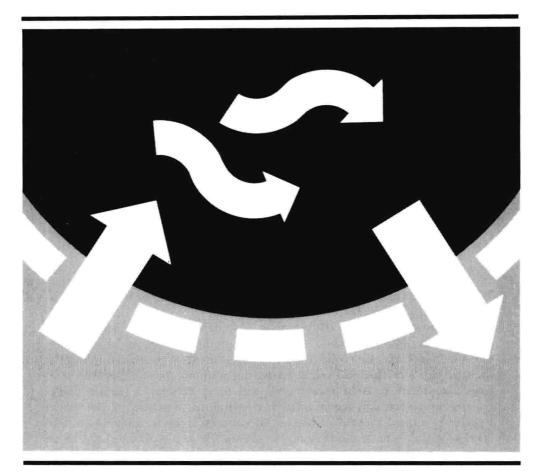
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## CHAPTER 1



## **INTRODUCTION**

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