



# Adsorption Refrigeration Technology

Theory and Application



RUZHU WANG | LIWEI WANG | JINGYI WU

Companion Website

WILEY

# **ADSORPTION REFRIGERATION TECHNOLOGY**

## **THEORY AND APPLICATION**

**Ruzhu Wang, Liwei Wang and Jingyi Wu**

*Shanghai Jiao Tong University, China*

**WILEY**

This edition first published 2014  
© 2014 John Wiley & Sons Singapore Pte. Ltd.

*Registered office*

John Wiley & Sons Singapore Pte. Ltd., 1 Fusionopolis Walk, #07-01 Solaris South Tower, Singapore 138628.

For details of our global editorial offices, for customer services and for information about how to apply for permission to reuse the copyright material in this book please see our website at [www.wiley.com](http://www.wiley.com).

All Rights Reserved. No part of this publication may be reproduced, stored in a retrieval system or transmitted, in any form or by any means, electronic, mechanical, photocopying, recording, scanning, or otherwise, except as expressly permitted by law, without either the prior written permission of the Publisher, or authorization through payment of the appropriate photocopy fee to the Copyright Clearance Center. Requests for permission should be addressed to the Publisher, John Wiley & Sons Singapore Pte. Ltd., 1 Fusionopolis Walk, #07-01 Solaris South Tower, Singapore 138628, tel: 65-66438000, fax: 65-66438008, email: [enquiry@wiley.com](mailto:enquiry@wiley.com).

Wiley also publishes its books in a variety of electronic formats. Some content that appears in print may not be available in electronic books.

Designations used by companies to distinguish their products are often claimed as trademarks. All brand names and product names used in this book are trade names, service marks, trademarks or registered trademarks of their respective owners. The Publisher is not associated with any product or vendor mentioned in this book. This publication is designed to provide accurate and authoritative information in regard to the subject matter covered. It is sold on the understanding that the Publisher is not engaged in rendering professional services. If professional advice or other expert assistance is required, the services of a competent professional should be sought.

Limit of Liability/Disclaimer of Warranty: While the publisher and author have used their best efforts in preparing this book, they make no representations or warranties with respect to the accuracy or completeness of the contents of this book and specifically disclaim any implied warranties of merchantability or fitness for a particular purpose. It is sold on the understanding that the publisher is not engaged in rendering professional services and neither the publisher nor the author shall be liable for damages arising herefrom. If professional advice or other expert assistance is required, the services of a competent professional should be sought.

***Library of Congress Cataloging-in-Publication Data***

Wang, Ruzhu.

Adsorption refrigeration technology : theory and application / Ruzhu Z. Wang, Liwei Wang, Jingyi Wu.

1 online resource.

Includes bibliographical references and index.

Description based on print version record and CIP data provided by publisher; resource not viewed.

ISBN 978-1-118-19746-2 (Adobe PDF) – ISBN 978-1-118-19747-9 (ePub) – ISBN 978-1-118-19743-1

(hardback) 1. Refrigeration and refrigerating machinery – Research. 2. Refrigeration and refrigerating machinery – Technological innovations. 3. Refrigeration and refrigerating machinery – Environmental aspects.

4. Adsorption. I. Wang, Liwei (Professor) II. Wu, Jingyi, Ph.D. III. Title.

TP492.5

621.5'7 – dc23

2014003757

Typeset in 10/12pt Times by Laserwords Private Limited, Chennai, India  
Printed and bound in Singapore by Markono Print Media Pte Ltd

ISBN: 978-1-118-19743-1

# **ADSORPTION REFRIGERATION TECHNOLOGY**



# About the Authors

**Ruzhu Wang** (R.Z. Wang) is a Professor of Institute of Refrigeration and Cryogenics at Shanghai Jiao Tong University. His major contributions are adsorption refrigeration, heat transfer of superfluid helium, heat pumps, CCHPs (cogeneration systems for cooling, heat, and power), and solar energy systems. He has published about 300 journal papers; about 200 of them are in international journals. He has written five books regarding Refrigeration Technologies. He was elected as CheungKong Chaired Professor in 2000 by the Ministry of Education (MOE) of China. Currently he is the vice president of the Chinese Association of Refrigeration, the vice chairman of the Chinese Society of Heat Transfer. Professor Wang was elected as one of the top 100 outstanding professors in Chinese universities in 2007. He was awarded as the model teacher of China in 2009. Professor Wang won second prize for the National Invention Award in 2010 on “Solar air conditioning and efficient heating units and their application,” and also received the second prize for the National Award for Education in 2009 for his ideas and successful practices on “Innovative, Globalization, and Research Learning” for talents education in the field of refrigeration.

**Liwei Wang** (L.W. Wang) is Professor of the Institute of Refrigeration and Cryogenics at Shanghai Jiao Tong University. Her research experience focuses on the conversion of low grade waste heat using the technology of adsorption, such as the adsorption refrigeration cycle, intensification of the heat and mass transfer performance of adsorbents, and adsorption cogeneration cycle for refrigeration and power generation. For her research work she received awards such as the National 100 Outstanding PhD Theses, IIR Young Researchers Award, Royal Society International Incoming Fellowship in the UK, and the EU Marie Curie International Incoming Fellowship.

**Jingyi Wu** (J.Y. Wu) is a Professor of the Institute of Refrigeration and Cryogenics at Shanghai Jiao Tong University. Her achievements are mainly in the utilization of low grade heat and cryogenics for aerospace. She has published over 130 papers and has led various research projects funded by National Natural Science Foundation of China (NSFC), Hi-Tech Research and Development Program, Aerospace Research Funding, and so on. As a main member, she won second prize at the National Invention Award (second prizes) in 2010 and the second prize in the National Award for Education in 2009.



# Preface

The supply and demand of energy determine the course of global development in every sphere of human activity. Finding sufficient supplies of energy to satisfy the world's growing demand is one of society's foremost challenges. Sorption refrigeration, which is driven by the low grade heat and provides the air conditioning and refrigeration effect, is paid more and more attention as one of the energy conversion technologies.

Sorption technology includes absorption and adsorption technology. The main differences between two types of technologies are the sorbents. The absorbents generally are liquid such as LiBr and  $\text{NH}_3$ , and the adsorbents are granular or compact solids, such as silica gel, zeolite, and chlorides. Compared with the absorption technology, the adsorption technology has the advantages of the wide choices of adsorbents for the wide scopes of driven temperatures for different heat sources, which generally ranges from 50 to 400 °C. The feature of solid adsorbents also makes it more feasible under the conditions with serious vibration. It doesn't need the rectifying equipments, nor does it have the problems of crystallization that can easily occur in absorption systems.

Adsorption refrigeration has two working processes. The first process is adsorption and refrigeration. In this process the adsorption heat releases cooling water or air to the heat sink and the pressure inside the adsorber decreases to a level lower than the evaporating pressure. The refrigerant evaporates and is adsorbed by the adsorbent under the function of pressure difference, and the evaporation process provides the refrigeration output. The second process is desorption and condensation. In this process the endothermic process of desorption is driven by the low grade heat. The desorbed refrigerant vapor is cooled by the heat sink and condensed in the condenser.

The earliest record of the phenomena of adsorption refrigeration was that AgCl adsorbed  $\text{NH}_3$ , which was discovered by Faraday in 1848. After that several refrigerators were developed for storing food and air conditioning. In the 1930s, the compression refrigeration technology was accelerated by technology innovations such as the discovery of Freon, the manufacture of a fully closed compressor, the application of compound refrigerants, and so on, and adsorption refrigeration could not compete with the CFCs (chlorofluorocarbons) system because of its low efficiency.

Since the late twentieth century, more and more research concentrated on sustainable development and the technology of adsorption refrigeration began to develop. There were two reasons for the fast development of sorption technologies: one is the need to solve the problems of energy shortage, which became more and more important since the worldwide energy crisis after the Middle East War during 1973. It takes about 7 million years to form petroleum



and current supplies have almost been used up after more than 200 years' of exploitation. The stock of coal is greater than petroleum, but it is also consumed quickly especially with increasing demand as people all over the world desire comfortable living standards. The recovery of the low grade heat is one of the main technologies that may overcome the increasing constraints related to energy utilization. Another reason is related to climate change caused by ozonosphere depletion. There is a common recognition by international academics that depletion of the ozonosphere is caused by CFCs, which are found in refrigerators, air conditioners, and heat pumps. The green refrigerants, which are common in sorption technologies, are now being focused on as a replacement for traditional compression refrigeration technology.

The main technologies on adsorption refrigeration which are being researched by academics are mainly advanced adsorbents, advanced cycles, and advanced design for refrigeration systems. For example, Professor Critoph in the UK has worked on adsorption refrigeration for over 20 years. He and his research team have developed the consolidated activated carbon needed for the refrigeration and thermal wave cycle for the high coefficient of performance. The research team in France, such as Spinner, Meunier, and Mauran have worked on chemisorption thermodynamics and developed IMPEX for refrigeration. The research team of Kashiwagi and Saha developed the silica gel–water adsorption chiller and proposed the multi-stage cycle; Lebrun studied the heat and mass transfer of adsorbers; Vasiliev developed the heat pipe type adsorbers; Aristov studied the composite adsorbents of silica gel and the thermodynamics of composite adsorbents; and the academics in Korea studied the heat and mass transfer performances of solidified adsorbent, and so on. But there are no books which have systematically summarized the technology of adsorption refrigeration although it has now been developed for over 150 years.

As researchers in Shanghai Jiao Tong University, P.R. China, we have researched adsorption refrigeration for over 20 years. The research aspects include adsorbents, adsorption working pairs, adsorption refrigeration cycles, and adsorption applications. In order to share our research experience with international academics we have summarized our achievements as well as other researchers' outcomes. In this book the history of the development of adsorption refrigeration, development of adsorbents, thermodynamic theories, design of adsorption systems, adsorption refrigeration cycles have been discussed step by step. The main objective of the book is to give the readers a comprehensive guide to the research on adsorption refrigeration.

Ruzhu Wang, Liwei Wang, Jingyi Wu  
2014

# Acknowledgments

We are grateful for the contributions from academics and students in our research team. They are: Dr. Z.Z. Xia and Dr. Z.S. Lu who contributed to the design and development of adsorption refrigeration systems, which were cited in the book; Prof. Y.J. Dai and Dr. X.Q. Zhai who contributed to the work on solar powered adsorption air conditioning; Dr. T.X. Li who contributed to the adsorption refrigeration cycles. Some of the contents of this book are from the theses of the Ph.D. students in the research team, and they are M. Li, T.F. Qu, Y.Z. Lu, S.G. Wang, Y.L. Liu, X.Q. Kong, X.Q. Zhai, H.L. Luo, K. Daou, D.C. Wang, K. Wang, Z.S. Lu, Y. Teng, and T.X. Li, et al. The research work of post doctors also was cited in the book, that is, the research work of Prof. W. Wang, S. Jiangzhou, Y.J. Dai, and R.G. Oliveira.

We also appreciate the support from the National Key Fundamental Research Program, National Natural Science Foundation of China (NSFC) for Distinguished and Excellent Young Scholars, NSFC Key Projects for Young Academics, and the Foundation from Science and Technology Commission of Shanghai Municipality, P.R. China.



# Nomenclature

$a$	Coefficient for the equilibrium reaction, coefficient in the van der Waals equation
$a_p$	The surface area per unit mass of adsorbent, $\text{m}^2/\text{kg}$
$a_v$	The surface area per unit volume of the adsorbent $\text{m}^2/\text{m}^3$
$A$	Coefficient in Clausius-Clapeyron equation
$A_0$	Dynamic coefficient
$A_{0b}$	The area of two back plates, $\text{m}^2$
$A_a$	Adsorbent cross-sectional area in the unit, $\text{m}^2$
$A_{adb}$	The heat transfer area of adsorber, $\text{m}^2$
$A_c$	The heat transfer area at the cooling side of the heat exchanger, $\text{m}^2$
$A_{eff}, A_{a,eff}$	Heat transfer area of heat exchanger at the solid adsorbent side, $\text{m}^2$
$A_{evf}$	The area at the fluid side of the heat pipe type evaporator, $\text{m}^2$
$A_f$	Heat transfer area of heat exchanger at the fluid side, $\text{m}^2$
$A_{fa}$	Internal surface area of the fin tube, $\text{m}^2$
$A_{fe}$	Anterior factor
$A_{fin}$	The area for the cross section of the fin, $\text{m}^2$
$A_{fm}$	The surface area of condensation pipe, $\text{m}^2$
$A_g$	Gas flow cross-sectional area in the unit, $\text{m}^2$
$A_m$	Heat transfer area of the metal wall at the adsorbent side, $\text{m}^2$
$A_{mr}$	Cross-sectional area of mass recovery channel, $\text{m}^2$
$A_{rx}, A_{ry}$	Constants in Mazet reaction models
$A_s$	The area of solar collector, $\text{m}^2$
$A_{seff}$	Effective collector area, $\text{m}^2$
$b$	Coefficient in the van der Waals equation
$B$	Parameter for the pore structure of the adsorbent
$c$	Concentration of adsorbate, $\text{kg}/\text{m}^3$
$c^*$	Equilibrium concentration corresponding to the adsorption capacity $x$ , $\text{kg}/\text{m}^3$
$c_i$	Concentration of the adsorbate on the surface of the adsorbent, $\text{kg}/\text{m}^3$
$C$	Constant in the Clausius-Clapeyron equation, specific heat, $(\text{J}/(\text{kg } ^\circ\text{C}))$
$C_{0\sim3}$	Coefficients in Tykodi models
$C_a, C_{pa}$	Specific heat of adsorbent, $\text{J}/(\text{mol K})$ , $\text{J}/(\text{kg } ^\circ\text{C})$
$C_{ca}$	Specific heat of composite adsorbent, $\text{J}/(\text{mol K})$ , $\text{J}/(\text{kg } ^\circ\text{C})$
$C_{Ha}$	Adsorbent heat capacity in the high-temperature adsorbent bed, $\text{J}/(\text{mol K})$ , $\text{J}/(\text{kg } ^\circ\text{C})$
$C_{hb}$	Specific heat of the liquid in the boiler, $\text{J}/(\text{mol K})$ , $\text{J}/(\text{kg } ^\circ\text{C})$

$C_{Lc}$	Specific heat of liquid refrigerant, J/(mol K), J/(kg °C)
$C_{Lv}$ , $C_{vg}$	Specific heat of refrigerant vapor, J/(mol K), J/(kg °C)
$C_m$ , $C_{pm}$	Specific heat of metal materials, J/(mol K), J/(kg °C)
$C_{mal}$	Specific heat of the aluminum, J/(mol K), J/(kg °C)
$C_{mcu}$	Specific heat of the copper, J/(mol K), J/(kg °C)
$C_{mh}$	Metal heat capacity of the heating boiler, J/(kg °C)
$C_p$	Isobaric specific heat, J/(mol K), J/(kg °C)
$C_{pb}$	The total thermal capacity, J/(mol K) or J/(kg °C)
$C_{pc}$ , $C_{pg}$	Isobaric specific heat of refrigerant vapor, J/(mol K), J/(kg °C)
$C_{pf}$	The thermal capacity of the fluid, J/(mol K), J/(kg °C)
$C_{pr}$ , $C_{pl}$	Isobaric specific heat of liquid refrigerant, J/(mol K) or J/(kg °C)
$C_{ps}$	The isobaric specific heat of solid material, J/(mol K), J/(kg °C)
$C_{pw}$	Thermal capacity of the metal walls, J/(mol K) or J/(kg °C)
$C_{ra}$	Proportional coefficient determined by evaporator type
$C_{vf}$	Specific heat at constant volume of the liquid refrigerant, J/(kg K)
$COP$	Coefficient of performance for refrigeration
$COP_{AC}$	COP for activated carbon adsorber
$COP_{carnot}$	COP for Carnot cycle
$COP_{hp}$	COP of heat pump
$COP_i$	Ideal COP
$COP_{int}$	COP for intermittent cycle
$COP_Z$	COP for zeolite adsorber
$d$	Distance, distance between molecules, diameter, m
$d_a$	The diameter of the adsorbent particles, m
$d_{ave}$	Average pore diameter, m
$d_e$	Equivalent diameter, m
$d_p$	Equivalent diameter of the solid particles, m
$d_{pi}$	Inlet diameter of the tube, Inner diameter of the pipe, m
$d_{po}$	Outer diameter of the pipe, m
$d_v$	Equivalent diameter for the flowing process of the vapor, m
$d_w$	The channel width, m
$D'$	The coefficient in D-A equation
$D_e$	Diffusion coefficient in the micropore, effective diffusion coefficient
$D_{go}$	Diameter of the outer glass tube, m
$D_i$	Effective diffusion coefficient, m <sup>2</sup> /s
$D_k$	Knudsen diffusion coefficient
$D_{ms}$	Mass diffusion coefficient of the fluid, m <sup>2</sup> /s
$D_s$ , $D_{so}$	Surface diffusion coefficient, m <sup>2</sup> /s
$e_{eff}$	Effective thickness of adsorbent, m
$e_{so}$	The internal energy for the solid adsorbent skeleton, kJ/kg
$E$	Specific adsorption power, J/mol
$E_a$	Activated energy for adsorption, J/mol
$E_d$	Activated energy for desorption, J/mol
$E_{ij}$	Thermal dispersion coefficient
$E_p$	Pseudo-activated energy, J/mol
$f$	The fugacity under the pressure of $p$ , Pa

$f_0$	The fugacity under the pressure of $p_s$ , Pa
$f_S$	The ratio between the area of airflow area and area of the cross-section area of wheel, $m^2/m^2$
$f_V$	Surface area of unit volume of adsorbent, $m^2/m^3$
$g$	Acceleration of gravity, $m/s^2$
$G$	Free enthalpy, J
$h$	Specific enthalpy, J/kg
$h_a, h_d$	Adsorption heat, desorption heat, kJ/kg
$h_{ev}$	The height for the evaporating section of the heat pipe, m
$h_f$	Specific enthalpy of the refrigerant liquid, J/kg
$h_r$	Specific enthalpy of the ammonia liquid at the condensation temperature, J/kg
$h_w$	The depth of the channels, m
$H$	Enthalpy, J
$H_a, H_d$	Adsorption heat, desorption heat, kJ
$H_{adb}$	The thickness (i.e., height) of the adsorbent bed, m
$\overline{H}_2$	Partial molar enthalpy, J/mol
$\overline{H}_g$	Molar enthalpy, J/mol
$H_{max}$	Maximum capillary height, m
$H_r$	Chemical reaction heat, J
$H_{st}$	Isobaric adsorption/desorption heat, kJ/kg
$I$	The solar radiation intensity, $W/m^2$
$I_0$	Direct sunlight intensity, $W/m^2$
$I_{ref}$	Reflected sunlight intensity from back plate, $W/m^2$
$J$	Heat flux, $W/m^2$
$k$	Coefficient in D-R equation
$k_1, k_2, k_3$	Stability constants
$k_F$	Mass transfer coefficient, $kg/(m^2 s)$
$k_{ij}$	The component of permeability tensor, $m^2$
$k_p$	Permeability of porous medium, $m^2$
$k_s$	Mass transfer coefficient inside the solid phase film, $kg/(m^2 s)$
$k_y$	Convection mass transfer coefficient, $kg/(m^2 s)$
$K$	The coefficient for D-R equation, equilibrium constant of the reaction, permeability ( $m^2$ )
$K_a$	Coefficient for the reaction rate in adsorption process, $1/(m^2 s)$
$K_d$	Coefficient for the reaction rate in desorption process, $1/(m^2 s)$
$K_F$	Mass transfer coefficient of the fluid side, m/s
$K_i$	The dynamic coefficient
$K_{ms}$	Coefficient of the mass transfer
$K_{md}$	Coefficient for the influence of chemical kinetics on the reaction
$K_n$	Knudsen diffusion rate
$K_r$	Reaction kinetic constant
$K_s$	The total mass transfer coefficient ( $kg/(m^2 s)$ ), permeability ( $m^2/s$ )
$K_s a_p$	Surface diffusion rate coefficient $1/s$
$K_v$	Net adsorption rate, $(kg/kg)/s$
$K_x$	Reaction coefficient in Iloeje's equation, $^{\circ}C/s$
$l$	Length, mass transfer scale, m

$l_{ah}$	Heat pipe height in the adsorbent bed, m
$l_{fin}$	The perimeter of the cross section, m
$L$	Latent heat of vaporization of refrigerant, kJ/kg
$L_a$	Adsorbent thickness along the direction of $L_y$ , m
$L_{ad}$	The length of adsorbent bed, m
$L_b$	The width of the adsorbent bed along the direction of $L_y$ , m
$L_{bw}$	Thickness of the wall, m
$L_B$	Unit lateral equivalent width, m
$L_c$	The condensation heat of the refrigerant in the condenser, kJ/kg
$L_e$	The evaporating heat of the refrigerant in the evaporator, kJ/kg
$L_{ev}$	The length of the evaporation section of the heat pipe type evaporator (m); the latent evaporation heat of the refrigerant (kJ/kg)
$L_{fin}$	The half distance between fins in the adsorption bed, m
$L_{hp}$	Evaporation latent heat of the fluid inside the heat pipe, kJ
$L_m$	Height of the heat medium along the direction of $L_z$ , m
$L_{pi}$	The length of the pipe, m
$L_{sat}$	Evaporation latent heat of the refrigerant at the temperature of $T_s$ , J/kg
$L_x, L_y, L_z$	Three coordinates, m
$L_{xt}$	The total length along the direction of $L_x$ , m
$\dot{m}$	Gas flow rate from a unit to the next unit, kg/s
$m$	Flow rate (kg/s, g/s)
$m_{am}$	Mass flow rate of ammonia, kg/s
$m_{air}$	The airflow rate, kg/s
$m_C$	The molar mass of $\text{CaCl}_2$ , 110.99
$m_e$	The mass flow rate of the vapor, kg/s
$m_f$	Volume flow rate of the fluid, $\text{m}^3/\text{s}$
$m_i$	Air flow through the unit cross-sectional area of wheel, $\text{kg}/(\text{m}^2\text{s})$
$m_{mr}$	The mass flow rate for the vapor in mass recovery phase, kg/s
$m_N$	Molar mass of $\text{NH}_3$ , 17
$m_{oil}$	Fuel quantity, kg/h
$m_{uA}$	Mass flow rate per unit area, $\text{kg}/(\text{m}^2 \text{ s})$
$m_w$	Mass flow rate of heating/cooling fluid, kg/s
$m_{water}$	Flow rate of the water, kg/s
$m_x, m_y$	Reaction order
$m_y$	Flow rate of the exhaust gas, kg/s
$M$	Mass, kg
$M_a$	The mass of adsorbent, kg
$M_{av}$	The adsorbent mass in unit volume, $\text{kg}/\text{m}^3$
$M_C$	The mass of $\text{CaCl}_2$ , kg
$M_{ca}$	The mass of composite adsorbent, kg
$M_{e0}$	Mass of the refrigerant in the evaporator under equilibrium conditions, kg
$M_{eqh}$	The mass of the working fluid in the liquid pumping boiler, kg
$M_{ev}$	The mass of the refrigerant in the evaporator, kg
$M_{ew}$	Mass of the refrigerant liquid inside the evaporator, kg
$M_g$	The mass of graphite, kg

$M_{ha}$	The mass of the working fluid in the fin tube of the adsorbent bed and in the liquid chamber, kg
$M_{hb}$	Total mass of the working fluid in the boiler, kg
$M_{hp}$	The initial mass of the working fluid in the liquid pumping boiler, kg
$M_{Ha}$	Adsorbent mass in the high temperature adsorbent bed, kg
$M_m$	The mass of support body in the unit volume, kg/m <sup>3</sup>
$M_{madb}$	Metal mass of the adsorbent bed, kg
$M_{mal}$	The mass of aluminum inside the adsorber, kg
$M_{mcu}$	The mass of the copper material inside the adsorber, kg
$M_{m,con}$	The metal mass of the condenser, kg
$M_{m,eva}$	The metal mass of the evaporator, kg
$M_{me}$	Mass of methanol desorbed from adsorber, kg
$M_{meva,cond}$	The metal mass of evaporator and condenser, kg
$M_{mh}$	Metal mass of the heating boiler, kg
$M_{phf}$	Mass of the liquid in the liquid pumping boiler that cannot be pumped into the adsorbent bed, kg
$M_r$	Reaction kinetic constant
$M_{Re}$	The function of the Reynolds number
$M_z$	Total mass of the working fluid filled into the heat pipe system, kg
$Ma$	Reaction dynamic coefficient for adsorption
$Md$	Reaction dynamic coefficient for desorption
$\bullet$	
$n$	The total molar flow rate, mol/s
$n$	Coefficient in D-A equation, coefficient for reaction equilibrium, reaction order
$n_2^s$	Molar adsorption quantity on the surface of solid adsorbent, mol/mol
$n_s$	Number of flow channels
$N$	Molar mass (mol), layer numbers of the glass cover
$N_g$	Molar adsorption quantity, mol/mol
$p$	Pressure, Pa
$p'$	Pressure on the metal chloride's surface, Pa
$p_{ae}, p_{ads}$	The pressure inside the adsorber at the end of the adsorption phase, Pa
$p_c$	Constrained pressure, Pa
$p_{de}, p_{des}$	The pressure of the adsorber at the end of the desorption phase, Pa
$p_{ea}$	Equilibrium pressure of adsorption state, Pa
$p_{ed}$	Equilibrium pressure of desorption state, Pa
$p_h$	Pressure of reaction interface, Pa
$p_i$	Pressure of the vapor reactant interface, Pa
$p_m$	The pressure of the system after the mass recovery, Pa
$P_{el}$	The electricity generation of the cogeneration system, W
$p_i$	Pressure inside pore, Pa
$PER$	Primary energy ratio
$Pr$	Prandtl number
$Pr_s$	Prandtl number of the media under the saturated temperature
$Pr_w$	Prandtl number of the media under the plate surface temperature of the heat exchanger
$q$	Heat flux density, W/m <sup>2</sup>



$q_{ads}$	Average differential adsorption heat, J/kg
$q_c$	Heat adsorbed by the adsorbent, J/kg
$q_{c,st}$	The cold storage quantity per unit mass of adsorbent, kJ/kg
$q_{h,st}$	The heat storage quantity per unit mass of adsorbent, kJ/kg
$q_{in}$	Endothermic heat, J/kg
$q_r$	The sum of the radiation, W/m <sup>2</sup>
$q_{reg}$	Required heat of the adsorbent bed without heat recovery process, J
$q_{reg}^*$	Heat recovered in a heat recovery process of the adsorbent bed, J
$q_{st}$	Isosteric heat, J/mol, J/kg
$Q$	Heat, J or kJ
$Q_{bind}$	The difference between the heat required for desorption $Q_{des}$ and the condensation heat $Q_{cond}$ , J or kJ
$Q_{cc}$	The sensible heat of the liquid refrigerant, J or kJ
$Q_{char}$	Charging heat, J or kJ
$Q_{chill}$	The heat at the refrigeration section of the heat pipe type evaporator, J or kJ
$Q_{eref}$	Cooling power generated by the evaporation of the refrigerant in evaporator, J or kJ
$Q_{evas}$	The sensible heat of liquid refrigerant in evaporator, J or kJ
$Q_{ew}$	The heat at the condensation section of the heat pipe type evaporator, J or kJ
$Q_{hg}$	Heat from the heat source, kJ
$Q_{hs}$	Heat quantity for convective heat transfer process, J or kJ
$Q_{h,st}$	The heat stored, J or kJ
$Q_{reg}$	Regenerative heat, J or kJ
$Q_{sens}$	Prerequisite energy to heat up the reactor to a required desorption temperature, J or kJ
$Q_{Hd}$	The desorption heat of the high temperature adsorber, J or kJ
$Q_{Hs}$	The synthezation heat of high temperature adsorber, J or kJ
$Q_{seff}$	Heat transformed from the actual solar radiation, J or kJ
$Q_{st}$	Isobaric adsorption heat, J or kJ
$Q_{sen}$	Sensible heat of the adsorber, J or kJ
$Q_{solar}$	Solar radiation, J or kJ
$r$	Radius, m
$r_{as}$	Ratio between expansion space and volume of adsorbent
$r_c$	Diameter of reaction surface, m
$r_g$	Radius of grain, m
$r_{hc}$	Heat recovery coefficient
$r_{sh}$	Shape factor of isothermal adsorption process of ideal adsorbent material
$R$	The universal gas constant, J/(mol K)
$R_0$	Thermal resistance of tube, (m <sup>2</sup> °C)/W
$R_f$	The thermal resistance of the fouling between the fluid and the metal wall, °C/W
$R_{go}$	The radius of the outer glass tube, m
$R_H$	Relative humidity, %
$R_i$	Thermal resistance of dirt, (m <sup>2</sup> °C)/W
$R_m$	The radius of metal tube, m
$R_p$	Average diameter of the adsorbent granules, m