

# 钱学森

## 力学手稿

Application of Schapig's Transformation  
Two Dimensional Flow

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钱学森

The equations of two dimensional motion of compressible fluids without rotation, assuming that the pressure is only a function of density, can be reduced to a single non-linear equation of the velocity potential. In the supersonic case, the problem is solved by Prandtl Meyer and Burgmann by means of the powerful method of characteristics. In the subsonic case, especially when the velocity is not too high, the velocity of sound is the velocity of small disturbances. The problem is then reduced to a linear equation of the velocity potential. This is the case of the flow of a fluid past a solid body. The disturbance superimposed on the uniform flow is sufficiently small. The velocity of sound is not too high. An example of this is the flow of a fluid past a thin airfoil due to the presence of stagnation points on the surface of the airfoil. But the presence of stagnation points on the surface of the airfoil makes the application of the linearized theory questionable at least near this region, because there the



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## 出版前言

2011年12月11日是西安交通大学杰出校友钱学森先生的百年诞辰。为缅怀钱学森学长,学习他的科学思想和卓越风范,展示其丰功伟绩和人格魅力,西安交通大学举办了“纪念钱学森诞辰100周年”系列活动:作为制片方之一,参与西部电影集团摄制传记故事片《钱学森》;与中央电视台合作,出品纪录片《实验班的故事——沿着钱学森走过的路》;扩建钱学森生平业绩展馆,向校内外开放;举办钱学森科学与教育思想研讨会;出版发行《钱学森力学手稿》、《钱学森年谱(初编)》、《钱学森第六次产业革命思想探微丛书》等。

钱学森先生在美国深造和工作期间留下大量珍贵手稿,这些手稿真实展示了钱学森先生博大精深的学识、开拓求实的精神和严谨奋进的作风,是钱老勇攀科学高峰和严谨治学的集中体现。这里,我们将部分原稿整理汇集成册,出版《钱学森力学手稿》,作为钱老百年诞辰的献礼。

《钱学森力学手稿》共10卷,包含两部分内容。第一部分是草稿,包括扁壳、球壳和圆柱壳屈曲分析的公式推导和数值演算。在研究圆柱壳轴压屈曲问题时,为了求得圆柱壳体的临界压力,在有关的五百多页草稿中,对多达二十多种可能的屈曲模

态逐一进行公式推演和数值计算,最终才找到满意的并在论文中采用的屈曲模态。仔细观察草稿中的数据列表,每个数字有效位数都长达八位,在手摇机械式计算机作为主要计算工具的年代,这串串数字凝聚着多少现今难以想象的艰辛劳动。

第二部分是手稿,以航空航天工程为核心,涵盖空气动力学、固体力学、火箭技术、工程控制论和物理力学等领域的部分学术论文手稿、打印稿和讲义。

《钱学森力学手稿》是在西安交通大学校领导的大力支持下,由西安交通大学航天航空学院沈亚鹏教授整理完成。图书出版过程中得到了西安交通大学党委宣传部、校友关系发展部、图书馆、航天航空学院等的积极协助,在此深表感谢。

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# Section 1

## *Two Dimensional Subsonic Flow of Compressible Fluids*



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### Summary

The basic concept of the present paper is to use <sup>a</sup>the tangent line to the adiabatic pressure-volume curve as an approximation to the curve itself. First, the general characteristics of such a fluid are shown. Then in Section I, a theory is developed <sup>for a fluid</sup> which in ~~main features~~ is similar to that of Demtch~~enko~~ and Busemann but is more general and can be applied to flow with velocity approaches<sup>ing</sup> that of sound. The theory is then applied to calculate the flow over elliptic cylinders. In Section II the work of H. Bateman is applied to this approximate adiabatic fluid and ~~two equations are given which express the relations between the velocity and the pressure distributions over a body in compressible flow to the velocity and the pressure distributions over the same body in incompressible flow. This~~ These equations are then used to predict the high speed characteristics of airfoils. ~~and results~~ the results <sup>obtained</sup> are essentially the same as ~~that~~ those obtained in Section I.

] The theory is put into a form, by which, knowing the incompressible flow over a body, the compressible flow over a similar body ~~near or to the first one in incompressible flow~~ can be calculated.



## TWO-DIMENSIONAL SUBSONIC FLOW

### OF COMPRESSIBLE FLUIDS

by

Hsue-shen Tsien  
California Institute of Technology

#### Introduction

Assuming that the pressure is a single-valued function of density only, the equations of two-dimensional irrotational motion of compressible fluids can be reduced to a single non-linear equation of the velocity potential. In the supersonic case, that is, in the case when ~~the~~ flow velocity is everywhere greater than that of local sound velocity, the problem is solved by Meyer & Prandtl and Busemann using the method of characteristics. The essential difficulty of this problem lies in the subsonic case, that is, in the case when ~~the~~ flow velocity is everywhere smaller than but near ~~to~~ the local sound velocity, because then the method of characteristics cannot be used. Glauert & Prandtl (Ref. 1) treated the case when the disturbance to the parallel rectilinear flow due to <sup>the</sup> presence of a solid body is small. They were then able to linearize the differential equation for the velocity potential and obtained an equation very similar to <sup>that</sup> ~~these~~ for ~~the~~ incompressible fluids. But there are usually stagnation points either in the surface of the body or in the field of flow, where the disturbance is no longer small. Hence, it is doubtful whether the linear theory can be applied to the flow near a stagnation point. On the same ground, the theory breaks down in ~~the~~ case of bodies whose dimension across the stream is not small compared with the dimension parallel to the stream.

To treat <sup>these</sup> ~~cases in which the body is blunt-nosed~~, Janzen and Rayleigh developed the method of successive approximations. This

method ~~was explained physically~~ <sup>was</sup> and put into a more convenient form by L. Poggi (~~Ref. 2~~) and P.A. Walther (~~Ref. 3~~). Recently C. Kaplan (Ref. 2) treated the case of flow over Joukowski airfoils and elliptic cylinders, using Poggi's method. However, the method is rather tedious and the convergent <sup>is</sup> very slow if the local velocity of sound is approached.

<sup>Begin</sup> Molenbroek (~~Ref. 5~~) and Tschapligin (~~Ref. 6~~) suggested the use of the magnitude of velocity  $w$  and inclination  $\beta$  of velocity <sup>to</sup> a chosen axis as independent variables, and were thus able to reduce the equation of velocity potential to a linear equation. This equation was solved by Tschapligin (~~Ref. 6~~) and ~~recently put into a more convenient form by F. Glauser and M. Glauser (Ref. 7).~~ The solution is essentially a series each term of which is a product of a hypergeometric function of  $w$  and a trigonometric function of  $\beta$ . The main difficulty in practical application of this solution is to obtain a proper set of boundary conditions in the plane of independent variables  $w, \beta$  and <sup>to</sup> put the solution in a closed form.

Tschapligin (~~Ref. 6~~) <sup>has</sup> shown that a great simplification of the equation in the hodograph plane results if the ratio of the specific heats of the gas is equal to -1. Since all real gases have their ratio of specific heats between 1 and 2, the value -1 seems without practical significance. It was Demtchenko (Ref. <sup>3</sup> 4) and Busemann (Ref. <sup>4</sup> 4) who made <sup>clarified</sup> clear the meaning of this specific value of -1. They found that this really means to take the tangent <sup>to</sup> of pressure-volume curve as an approximate <sup>ion</sup> to the curve itself. However, they limit themselves to use the tangent at the state of ~~rest~~ of the gas, <sup>corresponding to the stagnation point of flow</sup> about <sup>about</sup> Thus their theory can only be applied to flow with velocities up to

one half that ~~of the velocity~~<sup>of the sound</sup>. Recently, during a personal discussion, Th. von Karman suggested to the author that the theory can be generalized to use the tangent at the state of gas corresponding to the undisturbed parallel flow. Thus the range of usefulness of the theory can be greatly extended. This is carried out in the first <sup>section</sup> ~~part~~ of the present paper.

~~In the second part of this paper~~<sup>then</sup> this theory, based upon Demtchenko and Busemann's work, is applied to the case of flow over elliptic cylinders and the results compared with those of S.G. Hooker (Ref. ~~19~~<sup>5</sup>) and C. Kaplan (Ref. ~~2~~<sup>2</sup>). Furthermore, results calculated by Glauert~~1~~-Prandtl's linear theory are also included for comparison.

Recently, H. Bateman (Ref. ~~21~~<sup>6</sup>) demonstrated a remarkable reciprocity of two fields of flow of two fluids related by a certain point transformation. It will be shown in the <sup>second section</sup> ~~third part~~ of this paper that the flow of <sup>(and)</sup> incompressible fluid and the flow of compressible fluid approximated by the use of <sup>the</sup> tangent to adiabatic pressure-volume curve can be interpreted as such a point transformation. It is thus possible to obtain a solution for compressible flow whenever a solution of incompressible flow is known. ~~The difficulty, however, lies in~~

*This transformation from a flow of incompressible fluid to a flow of compressible fluid is found, however, essentially to be the same as that developed in Section I from Demtchenko's and Busemann's work. The only difference is*

~~body at any Mach's number whenever the low speed characteristics of the flow over the same body are known. The characteristics of the~~

incompressible flow can either be obtained by the well-known method of conformal mapping or by experiments. Due to the fact that practical aerodynamic engineers usually have the low speed characteristics at hand and that high speed data have to be obtained by use of a costly high speed wind tunnel the above mentioned relations are believed to be of considerable use to them.

In the fourth part of the paper, <sup>this</sup> the theory developed in ~~Part III~~ <sup>is used</sup> is applied to correlate airfoil data obtained by J. Stack (Ref. <sup>7</sup> ~~12~~) in the N.A.C.A. 24" high speed wind tunnel. The agreement with theory is found to be satisfactory. Then this theory is applied to predict the compressibility effect on the lift and moment of N.A.C.A. 4412 airfoil using experimentally determined pressure distribution over the same airfoil at low speed. The result is again compared with the more simple Glauert-Prandtl theory.

### Approximation to the Adiabatic Relation

~~Approximation to the Adiabatic~~

If  $p$  is the pressure,  $v$  is the specific volume and  $\gamma$  is the ratio of specific heats of a gas, the adiabatic relation  $p v^\gamma = \text{constant}$  is a curve in the  $p-v$  plane as shown in Fig. 1a. <sup>The</sup> conditions near the point  $(p_1, v_1)$  which corresponds to <sup>a</sup> the state of ~~fluid~~ undisturbed flow can be approximated by the tangent to the curve at that point. The equation of the tangent at this point can be written as

$$p_1 - p = C(v_1 - v) = C(s_1^{-1} - s^{-1}) \quad (1)$$

a state of undisturbed ~~fluid~~ flow -

where  $C$  is the slope of the tangent and  $\rho$  is the density of the fluid gas. Now the slope  $C$  must be equal to the slope of the curve at the point  $(p_1, v_1)$ , therefore,

$$C = \left( \frac{dp}{dv} \right)_1 = \left( \frac{dp}{d\rho} \frac{d\rho}{dv} \right)_1 = - \left( \frac{dp}{d\rho} \right)_1 \rho_1^2 = -a_1^2 \rho_1^2$$

where  $a_1$  is the sound velocity <sup>corresponding</sup> ~~corresponds~~ to the conditions  $p_1, v_1$ . Thus eq. (1) can be written as:

$$p_1 - p = a_1^2 \rho_1^2 \left( \frac{1}{\rho} - \frac{1}{\rho_1} \right) \quad (2)$$

This is an <sup>approximation</sup> ~~approximate~~ pressure-density <sup>to true</sup> ~~adiabatic~~ relation, and is shown in Fig. 1b with <sup>together the</sup> true adiabatic relation.

The <sup>generalized</sup> ~~Bernoulli's~~ theorem for compressible fluids is

$$\frac{1}{2} w_2^2 - \frac{1}{2} w_3^2 = \int_2^3 \frac{dp}{\rho} \quad (3)$$

where  $w$  is the velocity of the gas and the subscripts 2 and 3 denote two different states of the fluid. By substituting eq. (2) into eq. (3), the following relation is obtained:

$$\frac{1}{2} w_2^2 - \frac{1}{2} w_3^2 = \frac{1}{2} a_1^2 \rho_1^2 \left\{ \frac{1}{\rho_2} - \frac{1}{\rho_3} \right\} \quad (4)$$

[ of the fluid  
corresponding  
to the stagnation  
point of flow

Now if  $w_3 = 0$ ,  $w_2 = w$ ,  $\rho_3 = \rho_0$ , and  $\rho_2 = \rho$ , with the subscript 0 denotes the state of <sup>the</sup> rest, eq. (4) gives:

$$\frac{a_1^2 \rho_1^2}{\rho_0^2} + w^2 = \frac{\rho_1^2 a_1^2}{\rho^2} \quad (5)$$

If the square of sound velocity  $a^2$  is defined (as usually done) as the derivative of  $p$  with respect to  $\rho$ , eq. (2) gives:

$$a^2 \rho^2 = \frac{dp}{d\rho} \rho^2 = a_1^2 \rho_1^2 = \text{constant} \quad (6)$$

Therefore, eq. (5) can be written as:

$$\left(\frac{\rho}{\rho_0}\right)^2 = 1 - \frac{W^2}{a^2} \quad (7)$$

Similarly,

$$\left(\frac{\rho_0}{\rho}\right)^2 = 1 + \frac{W^2}{a_0^2} \quad (8)$$

It is interesting to notice that from eq. (8) the density decreases as velocity increases, as expected. Thus eq. (6) shows that the local velocity of sound increases as the velocity increases. This is just opposite to the real gas, because in the case of an adiabatic flow of a real gas it is well known that the temperature of gas decreases as the velocity of gas is increased, and thus the local sound velocity also decreases. However, in the present approximate theory, the ratio  $\frac{W}{a}$  or Mach's number still increases as the velocity increases, as can be seen by eq. (7). But this ratio only reaches the value unity when  $\rho = 0$ , or from eq. (8) when  $W = \infty$ . It is thus seen that the entire regime of flow is subsonic and thus the differential equation of the velocity potential ~~is of an incompressible~~ type. This is the reason why the complex representation of the velocity potential and stream function is possible for all cases, as will be shown in the following paragraphs. However, one should realize that the portion of the tangent that could be used as an approximation to the true adiabatic relation is that portion which lies in the first quadrant. Thus the upper limit of velocity for practical application of the theory ~~is~~ occurs

*] being proportional to the square root of the temperature*

*] is always of elliptic type, that is, always of same type as the differential equation of the velocity potential of incompressible fluids.*

at  $p=0$ . By using eqs. (17) and (18), this upper limit is found to be

$$\left(\frac{w}{w_1}\right)_{\max} = \frac{1}{\left(\frac{w_1}{a_1}\right)} \sqrt{\left(\frac{p}{a_1 s_1} + 1\right)^2 - \left\{1 - \left(\frac{w_1}{a_1}\right)^2\right\}} \quad (9)$$

[being the tangent point to the two adiabatic curves]

Since the point  $(p, s_1)$  lies on the ~~trans-adiabatic~~ curve, the relation  $a_1^2 = \gamma \frac{p}{s_1}$  can be used, and eq. (9) becomes:

$$\left(\frac{w}{w_1}\right)_{\max} = \frac{1}{\left(\frac{w_1}{a_1}\right)} \sqrt{\left(\frac{1}{\gamma} + 1\right)^2 - \left\{1 - \left(\frac{w_1}{a_1}\right)^2\right\}} \quad (10)$$

which is true for the adiabatic relation  $p s^{-\gamma} = \text{constant}$

This relation is plotted in Fig. 2. Since for most practical cases it is not likely that the ratio  $\left(\frac{w}{w_1}\right)$  will rise to values much higher than 2,  $p$  will remain positive, and this theory will give an approximate solution.

II with  $\gamma=1.4$

Velocity Method

Hodograph Method

Section I

If the flow is irrotational, there exists a velocity potential  $\phi$  such that

$$\frac{\partial \phi}{\partial x} = u, \quad \frac{\partial \phi}{\partial y} = v \quad (11)$$

where  $u, v$  are the components of  $w$  in the  $x$  and  $y$  direction, respectively. The equation of continuity,

$$\frac{\partial}{\partial x} \left( \frac{s}{s_0} u \right) + \frac{\partial}{\partial y} \left( \frac{s}{s_0} v \right) = 0$$

will be satisfied, if the stream function  $\psi$  is introduced such that

$$\frac{s}{s_0} u = \frac{\partial \psi}{\partial y}, \quad -\frac{s}{s_0} v = \frac{\partial \psi}{\partial x} \quad (12)$$



Now if the angle of inclination of the velocity  $w$  to the  $X$  axis is  $\beta$ , eqs. (11) and (12) give:

$$\begin{aligned} d\phi &= w \cos \beta dx + w \sin \beta dy \\ d\psi &= -w \frac{s_0}{s} \sin \beta dx + w \frac{s_0}{s} \cos \beta dy \end{aligned} \quad (13)$$

Solving for  $dx$  and  $dy$ ,

$$\begin{aligned} dx &= \frac{\cos \beta}{w} d\phi - \frac{\sin \beta}{w} \frac{s_0}{s} d\psi \\ dy &= \frac{\sin \beta}{w} d\phi + \frac{\cos \beta}{w} \frac{s_0}{s} d\psi \end{aligned} \quad (14)$$

So long as the correspondence between the physical ~~plane~~ and hodograph plane is one to one, or mathematically  $\frac{\partial(x, y)}{\partial(w, \beta)} \neq 0$ , ~~one can~~ one can express  $x$  and  $y$  as functions of  $w, \beta$ , and  $\phi$  and  $\psi$  as functions of  $w, \beta$ . Thus,

$$d\phi = \phi'_w dw + \phi'_\beta d\beta \quad (15)$$

$$d\psi = \psi'_w dw + \psi'_\beta d\beta$$

where primes indicate the derivative with respect to variables

indicated as subscripts. Now substituting eq. (15) into eq. (14), the following expressions one has: for  $dx$  and  $dy$  are obtained:

$$\begin{aligned} dx &= \left( \frac{\cos \beta}{w} \phi'_w - \frac{\sin \beta}{w} \frac{s_0}{s} \psi'_w \right) dw + \left( \frac{\cos \beta}{w} \phi'_\beta - \frac{\sin \beta}{w} \frac{s_0}{s} \psi'_\beta \right) d\beta \\ dy &= \left( \frac{\sin \beta}{w} \phi'_w + \frac{\cos \beta}{w} \frac{s_0}{s} \psi'_w \right) dw + \left( \frac{\sin \beta}{w} \phi'_\beta + \frac{\cos \beta}{w} \frac{s_0}{s} \psi'_\beta \right) d\beta \end{aligned} \quad (16)$$

Since the left-hand side of eqs. (16) are exact differentials, one can apply the reciprocity relation, and obtain therefore the following expressions one has: for  $dx$  and  $dy$  are obtained:

$$\begin{aligned} \frac{\partial}{\partial \beta} \left( \frac{\cos \beta}{w} \phi'_w - \frac{\sin \beta}{w} \frac{s_0}{s} \psi'_w \right) &= \frac{\partial}{\partial w} \left( \frac{\cos \beta}{w} \phi'_\beta - \frac{\sin \beta}{w} \frac{s_0}{s} \psi'_\beta \right) \\ \frac{\partial}{\partial \beta} \left( \frac{\sin \beta}{w} \phi'_w + \frac{\cos \beta}{w} \frac{s_0}{s} \psi'_w \right) &= \frac{\partial}{\partial w} \left( \frac{\sin \beta}{w} \phi'_\beta + \frac{\cos \beta}{w} \frac{s_0}{s} \psi'_\beta \right) \end{aligned} \quad (17)$$

Carrying out these differentiations and simplifying with the aid of eq. (7), eq. (17) gives:

$$\begin{aligned} -\frac{\sin\beta}{w} \phi'_w + \frac{\cos\beta}{w} \frac{s_0}{s} \psi'_w &= -\frac{\cos\beta}{w^2} \phi'_\beta + \frac{\sin\beta}{w^2} \frac{s_0}{s_0} \psi'_\beta \\ \frac{\cos\beta}{w} \phi'_w - \frac{\sin\beta}{w} \frac{s_0}{s} \psi'_w &= -\frac{\sin\beta}{w^2} \phi'_\beta - \frac{\cos\beta}{w^2} \frac{s_0}{s_0} \psi'_\beta \end{aligned} \quad (18)$$

Solving for  $\phi'_w$  and  $\psi'_\beta$ ,

$$\begin{aligned} \phi'_w &= -\frac{s}{s_0} \frac{1}{w} \psi'_\beta \\ \phi'_\beta &= \frac{s_0}{s} w \psi'_w \end{aligned} \quad (19)$$

Eq. (19) can be further simplified by introducing a new variable  $\omega$ , such that

$$d\omega = \frac{s}{s_0} \frac{dw}{w} \quad (20)$$

Then eq. (19) becomes:

$$\begin{aligned} \phi'_{\omega} &= -\psi'_{\beta} \\ \phi'_{\beta} &= \psi'_{\omega} \end{aligned} \quad (21)$$

This can be easily recognized as ~~the~~ <sup>the</sup> Riemann-Cauchy differential equations, and thus  $\phi + i\psi$  must be an analytic function of  $\omega - i\beta$ . However, for convenience of calculation, another new set of independent variables  $U = W \cos\beta$ ,

$$V = W \sin\beta \quad \text{are introduced where} \quad W = a_0 e^{\omega}$$

Then eq. (21) can be written as:

$$\begin{aligned} \frac{\partial \phi}{\partial U} &= \frac{\partial \psi}{\partial (-V)} \\ \frac{\partial \phi}{\partial (-V)} &= -\frac{\partial \psi}{\partial U} \end{aligned} \quad (22)$$

~~and also~~ By integrating eq. (20),

$$W = \frac{2a_0 w}{\sqrt{a_0^2 + w^2}} + a_0 \quad (23)$$